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1 Purpose of this Document

This document explains design decisions that lead to the standardized applications interfaces relevant to the Powertrain Domain.

The sensor actuator pattern described in this document is not specific to the powertrain domain but can be applied to other domains too, e.g. the chassis domain.

<u>NOTE:</u> If any information in diagrams or text (or conclusions drawn from them) conflict with the information in [3] or [3b] and this is not explicitly mentioned the information in [3] or [3b], resp., should be regarded as definitive.



2 References

- [1] SW-C and System Modeling Guide AUTOSAR_TR_SW-CModelingGuide
- [3] XML Specification of Application Interfaces AUTOSAR_MOD_AISpecification
- [3b] Application Interfaces Examples AUTOSAR_MOD_AISpecificationExamples
- [4] Explanation of Application Interfaces of the Chassis Domain AUTOSAR_EXP_AIChassisExplanation
- [5] Unique Names for Documentation, Measurement and Calibration: Modeling and Naming Aspects including Automatic Generation AUTOSAR_TR_AIMeasurementCalibrationDiagnostics
- [6] Software Component Template AUTOSAR_TPS_SoftwareComponentTemplate
- [7] Standardization Template AUTOSAR_TPS_StandardizationTemplate
- [8] ANTLR parser generator V3 http://www.antlr.org
- [9] Virtual Function Bus AUTOSAR_EXP_VFB
- [10] Glossary AUTOSAR_TR_Glossary
- [11] AIDesignPatternCatalogue AUTOSAR_TR_AIDesignPatternCatalogue



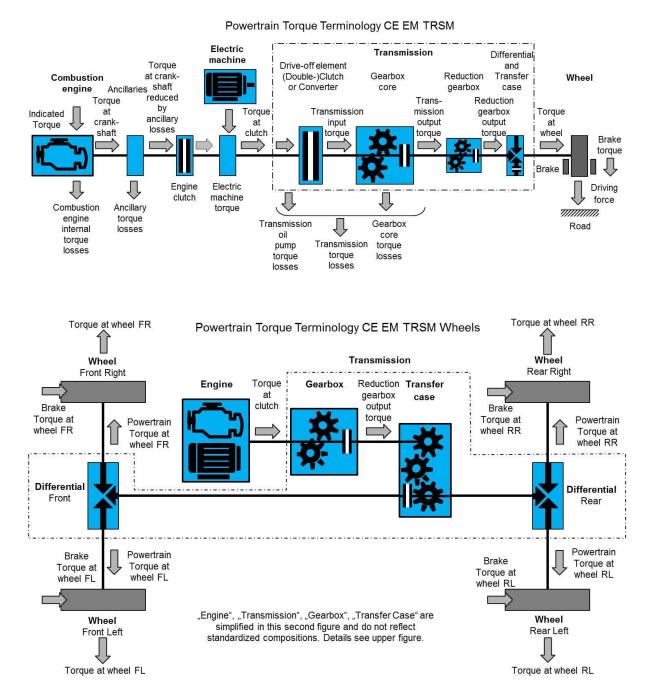
3 Description of Terms and Concepts

3.1 Abbreviations

For abbreviations used in this document please refer to the keyword list in [3] (as .arxml).

Additionally, please also refer to [10] for explanation of commonly used terms and abbreviations within AUTOSAR.

3.2 Terminology – Torque within the Powertrain Domain





Sign definition for torque at clutch / torque at wheels:

Positive value means that torque is transmitted from the engine to the drivetrain / from the powertrain to the wheels.

Negative value means that torque is transmitted from the drivetrain to the engine / from the wheels to the powertrain.

Zero means that no torque is transmitted between engine and drivetrain / between wheels and powertrain.

Ancillary torque losses:

Ancillary torque losses are losses with influence on "torque at crankshaft reduced by ancillary losses" and "Torque at clutch" caused by e.g. alternator, airconditioning, power steering.

Consideration of the inertia in torque signals:

Torque influence of inertia is not considered, viz.

Torque effect caused by the inertia is not permitted to be considered additionally on engine side, viz.

- for an increasing engine speed, the indicated torque is not permitted to be increased by the torque which is necessary for the positive speed gradient, respectively
- for a decreasing engine speed, the indicated torque is not permitted to be decreased by the torque which is necessary for the negative speed gradient.

Engine Clutch:

For Hybrid Systems an additional clutch can be present between combustion engine and electric machine.

3.3 Terminology – Fast and Slow Torque Requests

Many torque request interfaces have the additional descriptors "Fast" or "Slow". These descriptors are relevant to gasoline spark ignition engines, whose torque output can be modified by means of throttle angle (and hence air mass) and ignition timing. In general, the torque output responds slowly to changes in throttle angle due to fluid dynamics in the manifold and cylinder head. The reaction to ignition timing changes is almost instantaneous, especially at higher engine speeds.

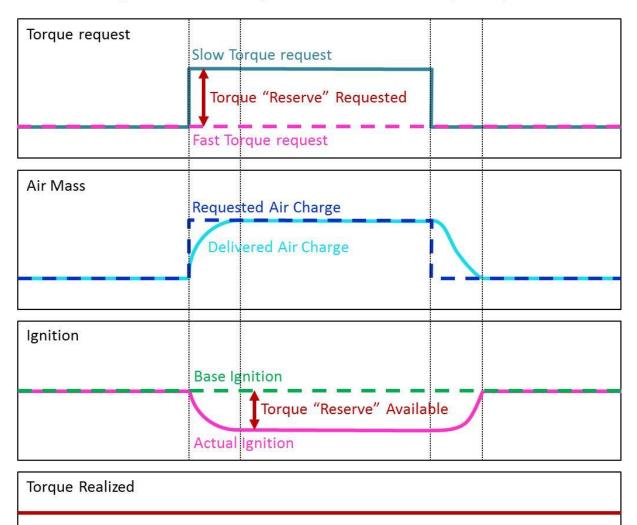
"Fast" refers to the "immediate" / "instant" torque request, typically achieved by ignition timing.

"Slow" refers to the longer term or "torque reserve" request, usually the input to throttle control.

Note that a gasoline engine running at optimum ignition timing cannot **increase** torque quickly as the throttle is the only means for the increase. However, preemptively opening the throttle and running with retarded ignition to maintain the the original (lower) torque allows the torque to be increased quickly by ignition a short time in the future. This operation is usually achieved by setting the "Slow" torque request to be greater than the "Fast" torque request to provide this "torque reserve", allowing the torque to be rapidly increased by increasing the "Fast" request.



Torque Reserve concept with Fast and Slow torque requests



Torque Reserve concept with Fast and Slow torque requests

For conventional diesel engines only the fast torque interfaces are relevant. However, future diesel engines could have the possibility to use both fast and slow torque interfaces.



3.4 Typical combinations of signals for transmission shift intervention

Basically, there are two different possibilities to transmit a transmission torque at clutch request:

A) Torque request via <u>one</u> torque signal, which can transmit increasing and reducing torque requests *Transmission: Request for Torque at Clutch for Shift Intervention* in combination with a Request for implementation type, which defines if the request on the torque signal is an increasing or a reducing one, an absolute or a relative one *Transmission: Implementation Type of Request for Torque at Clutch for Shift Intervention vention*

or

- B) Request via two torque signals and one signal for implementation type:
 - One torque signal defines the "maximum torque", which reflects the upper limit of the allowed torque. This is used to request a reducing torque intervention, *Transmission: Request for Maximum Torque at Clutch for Shift Intervention on Fast Path,*
 - The other torque signal defines the "minimum torque", which reflects the lower limit of the requested torque. This is used to request an increasing torque intervention, *Transmission: Request for Minimum Torque at Clutch for Shift Intervention on Fast Path,*
 - The signal for implementation type defines, if the request is an absolute one (request to decrease/increase torque TONm) or a relative one (request to decrease/increase torque BYNm), *Transmission: Implementation Type of Request for Maximum and Minimum Torque at Clutch for Shift Intervention on Fast and Slow Path.*

Both possibilities may be accompanied project specific by extended or more detailed requirements as, for example,

- Intervention mode, which requests that the intervention has to be implemented by ignition angle and/or air and/or cylinder cut off.
- A torque reserve (which has no influence to "torque at clutch") and its request for implementation type. These two signals prepare the possibility for an upcoming fast increasing torque intervention on gasoline engines.



Request for shift intervention concerning possibility A:

Transmission: Request for Torque at Clutch for Shift Inter- vention	Request for an increasing or reducing intervention	
	Request for implementation type: absolute minimum / absolute maximum / absolute exact /	
Transmission: Implementation Type of Request for Torque at Clutch for Shift Intervention	relative Definition of "absolute": Reduce / increase torque to Nm. Definition of "relative": Reduce / increase torque by Nm.	- Signals initiate a real torque modification
Transmission: Intervention Mode Request for Torque at Clutch for Shift Intervention	Request for intervention mode: ignition angle and cylinder cut-off / ignition angle / ignition angle and air / air / cylinder cut-off / cylinder cut-off and air	
Request for shift interv	ention concerning possibility B:	I
Transmission: Request for Maximum Torque at Clutch for Shift Intervention on Fast Path	Request for a fast reducing intervention	
Transmission: Intervention Mode Request for Maximum Torque at Clutch for Shift Inter- vention on Fast Path	Request for intervention mode of the "Request for a fast reducing intervention": ignition angle and cylinder cut-off / ignition angle / cylinder cut-off.	
Transmission: Request for Minimum Torque at Clutch for Shift Intervention on Fast Path	- Request for a fast increasing intervention	
Transmission: Implementation Type of Request for Maximum and Minimum Torque at Clutch for Shift Intervention on Fast and Slow Path	Request for implementation type: absolute / relative Definition of "absolute": Reduce / increase torque to Nm. Definition of "relative": Reduce / increase torque by Nm.	- Signals initiate a real torque modification
Transmission: Request for Maximum Torque at Clutch for Shift Intervention on Slow Path	- Request for a slow reducing intervention	
Transmission: Request for Minimum Torque at Clutch for		

Request for a slow increasing intervention

Request for a torque reserve as add on for possibilities A and B:

Transmission: Request for Torque Reserve at Clutch

Shift Intervention on Slow Path

Transmission: Implementation Type of Request for Torque Reserve at Clutch Predicted non-binding information about an upcoming fast increasing engine intervention. Initiates a torque reserve. Request for implementation type of the torque reserve:

Signals must not initiate a real torque modification! Consideration has to be neutral for torque at clutch!



3.5 Overview of AUTOSAR torque application interfaces

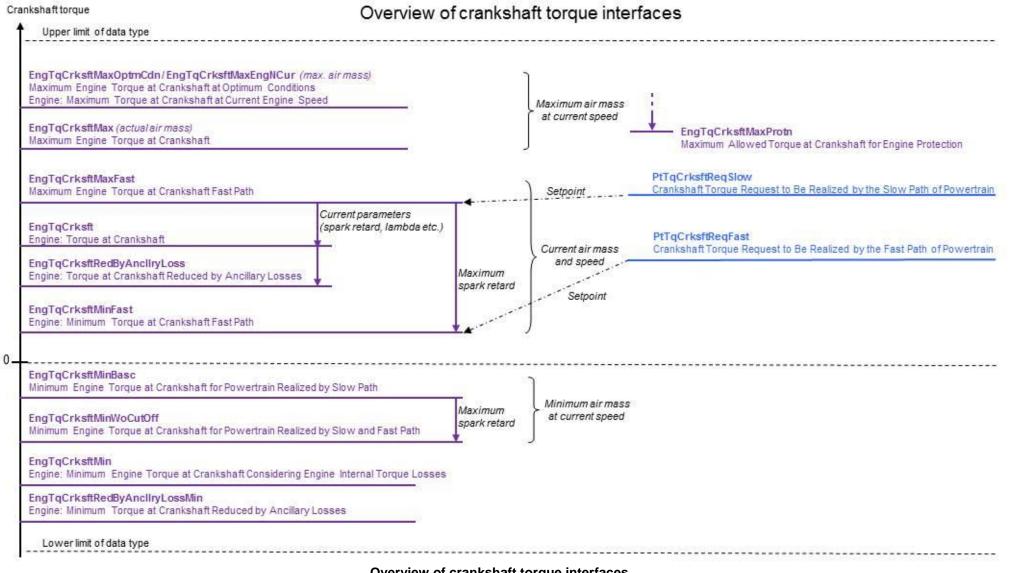
Legend:

<shortName of Port>

<longName of Port>

Arrows represent the <u>direction</u> of the requirement (increase/ decrease of torque). Horizontal lines represent the <u>target level</u> of the requirement.

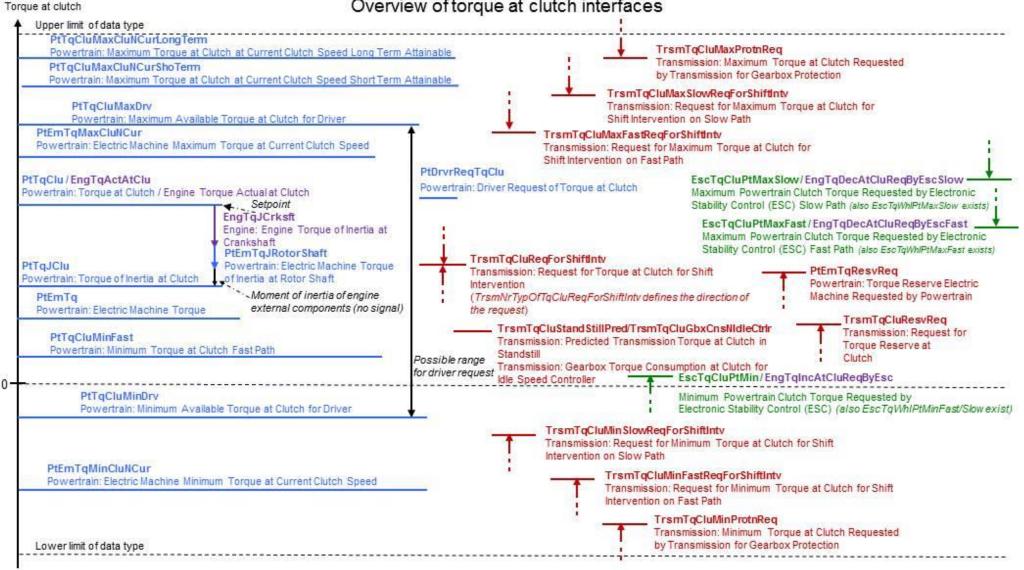




Overview of crankshaft torgue interfaces

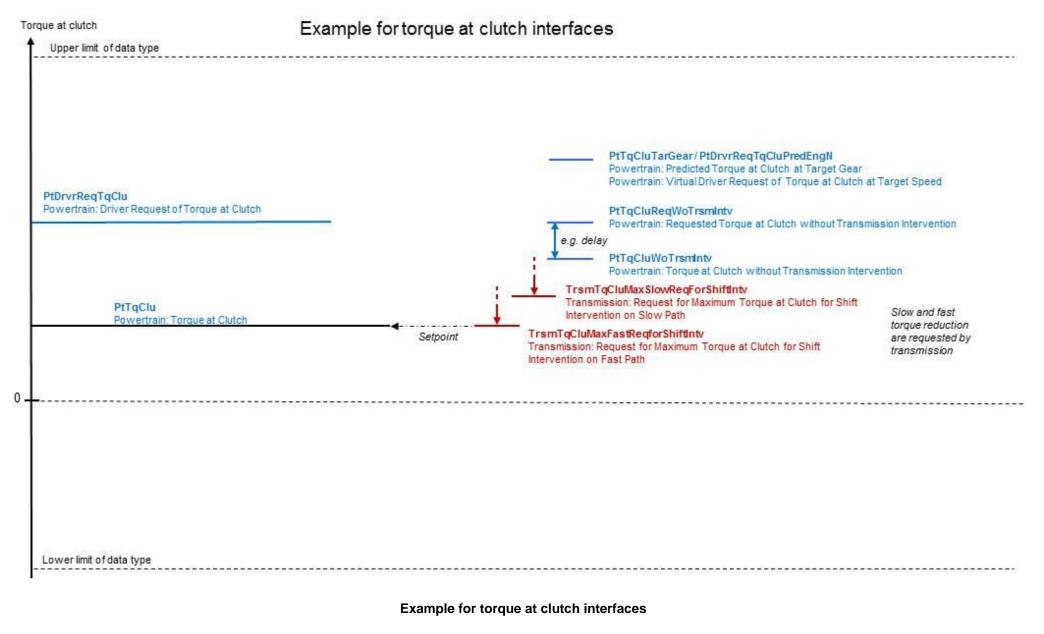
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Overview of torque at clutch interfaces



Overview of torgue at clutch interfaces







Explanation of Application Interfaces of the Powertrain Engine Domain AUTOSAR CP R20-11

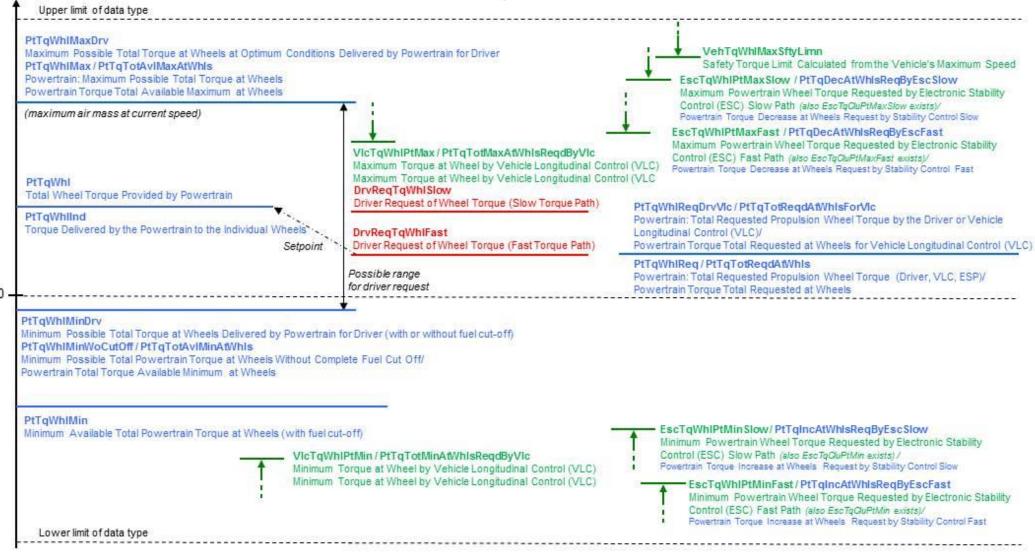
que at wheel	Ov	erview of torque loss interfa	aces	
Upper limit of data type				
AltTq Alternator Mechanical Torque (Load) AltTqReq Requested Mechanical Torque for Alternator (Ger PtTqMaxAlt Powertrain: Maximum Allowed Mechanical Load for Engine Crank Shaft.	AND THE PERSON AND A DECEMPERATION OF	AcTq Mechanical Torque for A/C Compressor AcTqReq Requested Mechanical Torque for A/C Comp PtTqMaxAc Powertrain: Maximum Allowed Torque for A	New Course and	TrsmTqLoss Transmission: Transmission Torque Losse
Lower limit of data type	Steering Load Hydraulic SteerTqReg / SteerLoa Requested Hydraulic Pov Steering Load Hydraulic PtTqMaxSteer	Steering Load at Engine Crank Shaft/ adHydReq wer Steering Load at Engine Crank Shaft/	TrsmTqOilPmpRe Requested Transmi PtTqMaxTrsmOilP	ission Oil Pump Load

Overview of torque loss interfaces



Torque at wheel

Overview of torque at wheel interfaces

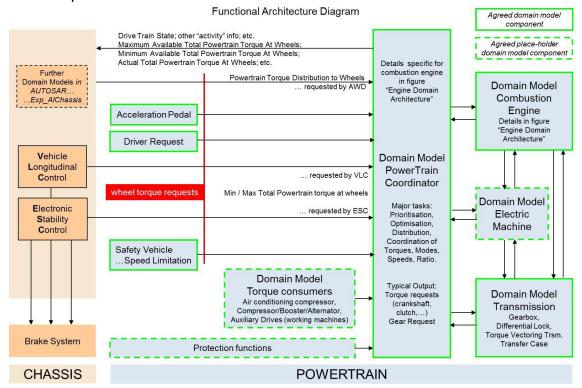


Overview of torque at wheel interfaces



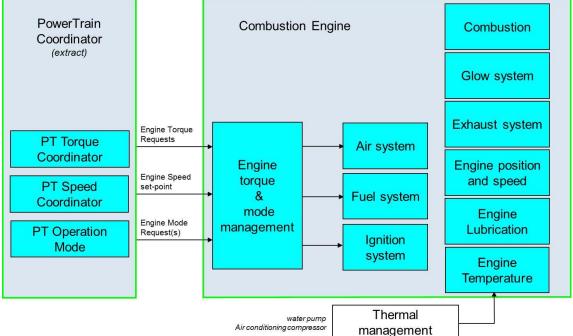
4 Architecture Overview

The following figures give an overview of the domain or functional architecture. They not necessarily give a complete picture but show the most relevant interconnections and components.



Functional Architecture Diagram

Engine Domain Architecture Diagram



Engine Domain Architecture Diagram



5 Description of Exemplary Software Components

For being able to use and understand the standardized application interfaces a typical domain architecture was used as basis for demonstrating the signal flow. The components of this example domain architecture are described in the following.

5.1 Powertrain Coordinator – PTC (PtCoorr)

This composition includes all functions that coordinate the operation of the Powertrain, including:

Powertrain operation mode – management of states of all actuators (e.g. combustion engine, clutch(es), transmission, electric motors, etc.), including engine start / stop management (conventional & hybrid Powertrains).

Powertrain torque coordination – Torque coordination at Powertrain (PT) level, torque prioritisation, torque distribution for realization at at PT level, torque reserve request for the PTC, pre-coordination of driveability functions for hybrids, Powertrain driveability filters, determination of total Powertrain losses for torque calculation, wheel torque calculation (min, max, consolidated), torque at clutch calculation (min, max, consolidated), torque at clutch calculation (min, max, consolidated), transformation of torque set point from wheel torque to torque at clutch, transformation of torque set point from torque at clutch to torque at crankshaft, control/coordination of auxiliary drivers/actuators.

Powertrain speed coordination – Maximum speed limitation coordination (for protection of all PT components from damage from over speed) and coordination of idle speed / engine speed set point requests from all sources, e.g. transmission.

Powertrain ratio coordination – all transmission ratio set point logic. Note that realization of ratio set point is carried out by transmission system, not PTC.

5.2 Transmission System (Trsm)

This composition includes all functions of the transmission system, including: *Transmission system coordination* – Determines the torque and speed ratio over transmission, converter and differential, including the calculation of torque losses in the transmission system. Coordinates mechanical protection of the Drivetrain (gearbox, driveshafts, etc.), including calculation of torque limitation.

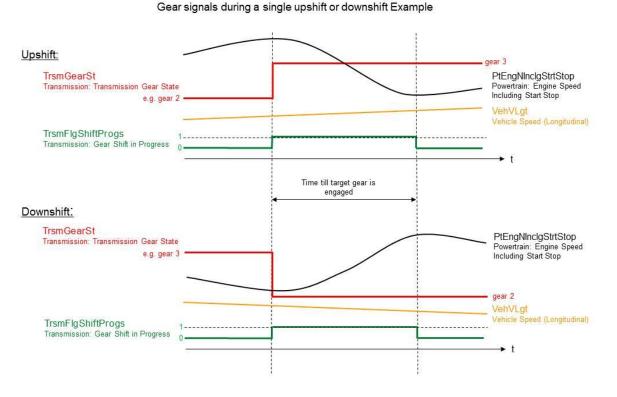
For manual transmission, this function includes the determination of the current gear and clutch status.

Transmission – Management of particular states in the transmission, including shift transition, driving off situation, creeping mode etc.. In case of shift transition, this functionality calculates torque requests to optimise the transition.

Control of transmission actuators to adjust the gear to the target gear (or to adjust the gear ratio to the target gear ratio in case of CVT). Gear ratio means the theoretical / physical ratio belonging to each gear and not any actual measured value. Control of gearbox countershaft (low/higher range) actuators is not included.

Calculates the torque gain of a hydrodynamic converter and the torque required to the converter input side in idle, etc. and controls clutch or converter actuators. All functionality related to the protection of the transmission, including calculation of torque limitation, measurement or calculation of gearbox oil temperature, etc., and calculation of requests to other systems.





Gear signals during a single upshift or downshift Example

Drivetrain Torque Distribution (DtTqDibtn) Differential Lock – All functionality related to the differential(s), which manage the torque distribution between left and right wheels, for example locking of the differential. Does not include the calculation of the distribution set point.

Drivetrain Torque Distribution (DtTqDibtn) Transfer Case – All functionality related to the transfer case, which manages the torque distribution between front and rear wheels. Does not include the calculation of the distribution set point.

Drivetrain Torque Distribution (DtTqDibtn) Torque vectoring axle transmission – All functionality related to active distribution of powertrain torque to all four wheels individually. Does not include the calculation of the distribution set point.

For additional information on Drivetrain Torque Distribution (DtTqDibtn) please also refer to [4].

5.3 Combustion Engine (CmbEng)

This composition includes all functions directly related to the operation and control of the vehicle's combustion engine. The following sections, 5.3.1 to 5.3.3 inclusive, define the components as a result of Combustion Engine functionality decomposition agreed to date.

5.3.1 Engine Speed and Position (EngSpdAndPosn)

Functions that provide all parameters linked to engine shaft position and speed, including the synchronisation on between crankshaft and camshaft.



Crankshaft and camshaft signal acquisition.

Calculation of the engine position.

Calculation of the relative camshaft position for systems with variable valve timing and/or lift.

Related diagnosis and plausibility checks.

5.3.2 Engine Torque Mode Management (EngTqModMngt)

Includes calculation of engine torque set point, realization of that set point (coordination of air / fuel / ignition, etc.), determination of consolidated engine torque, control of engine speed (idle / off-idle / limitation), and management of engine modes (including overall mode, modes for realization of engine start & stop, and combustion modes).

5.3.3 Combustion Engine: Miscellaneous (CmbEngMisc)

Combustion Engine Misc gathers together miscellaneous engine interfaces. In general, these are common data required for correct operation of the engine (engine temperature, ambient air pressure and battery voltage) or required for fail-safe actions (crash status). The way in which these interfaces are used is not standardized. In future AUTOSAR releases, it is likely that these interfaces may be moved to different (more appropriate) provider or receiver components / compositions.



5.3.4 Combustion Engine: Timing- and Accuracy requirements on the signal "Powertrain: Engine Speed Including Start Stop", necessary for transmission.

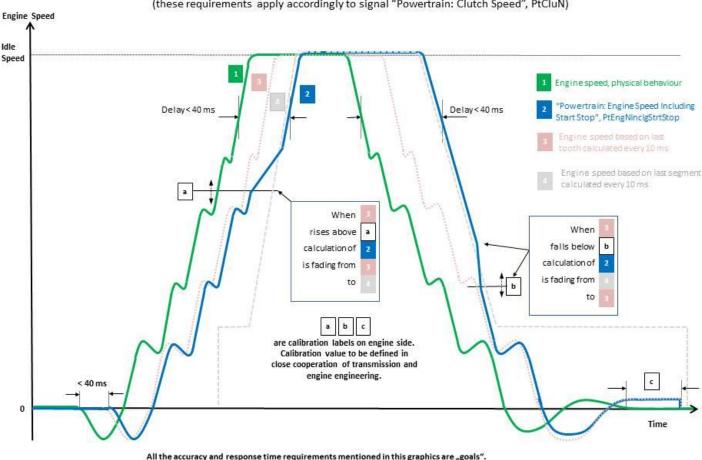
Timing- and Accuracy requirements to signal "Powertrain: Engine Speed Including Start Stop", PtEngNInclgStrtStop (these requirements apply accordingly to signal "Powertrain: Clutch Speed", PtCluN) "real/physical" engine speed (average over a crankshaft angle 720° divided by number of **Engine Speed** cylinders) signal "Powertrain: Engine Speed Including Start permissible offset +/- 6 rpm, permissible spread +/- 10 rpm Stop", PtEngNinclgStrtStop Accuracy signal "Powertrain: Engine Speed Including Start Stop", PtEngNInclgStrtStop max. at max. engine speed: permissible offset +/- 6 rpm, permissible spread +/- 10 rpm permissible offset at 800 rpm: permissible offset +/- 6 rpm, +/-6rpm, permissible spread +/- 3 rpm permissible spread 800 rpm +/-2 rpm Idle Speed Delay of signal "Powertrain: Engine Speed at engine speed < 800 rpm: permissible offset +/- 6 rpm, Including Start Stop", PtEngNInclgStrtStop<40 ms permissible spread +/- 2 rpm throughout the entire operating range 0 Time

All the accuracy and response time requirements mentioned in this graphics are "goals". Project specific requirements have to be defined in close cooperation of transmission and engine engineering.

Engine Speed Including Start Stop Accuracy Requirements



Additional requirements for timing and accuracy on the signal "Powertrain: Engine Speed Including Start Stop" during engine start and shut down:



Engine Speed Including Start Stop Composition

(these requirements apply accordingly to signal "Powertrain: Clutch Speed", PtCluN)

Engine Speed Including Start Stop Composition

Project specific requirements have to be defined in close cooperation of transmission and engine engineering.

Legend:

Calculation of "Powertrain: Engine Speed Including Start Stop" on the engine side:

Combination of

"Engine speed based on last segment calculated every 10 ms" and

"Engine speed based on last tooth calculated every 10 ms".

For a high engine speed, greater than the upper hysteresis limit "a", signal "Powertrain: Engine Speed Including Start Stop" is "Engine speed based on last segment calculated every 10 ms".

For a low engine speed, less than the lower hysteresis limit "b", the signal "Powertrain: Engine Speed Including Start Stop" is "Engine speed based on last tooth calculated every 10 ms".

The transition between a high engine speed and a low engine speed is defined by a hysteresis with an upper threshold "a" and a lower threshold "b".



If the engine speed increases to a value greater than above "a", then the number of teeth used for the speed calculation is incremented up to segment length using a defined step width.

The incrementation is done depending on the time after crossing "a". If the engine speed decreases to a value less than "b", then the number of teeth used for the speed calculation is decremented down to one using a defined step width.

The incrementation is done depending on the time after crossing "b".

The input for the hysteresis is "Engine speed based on last tooth calculated every 10 ms".

At a negative engine speed "Powertrain: Engine Speed Including Start Stop" is calculated based on one tooth.

Additional requirements on "Powertrain: Engine Speed Including Start Stop" in the range of 0 rpm up to an idle speed are necessary to ensure comfortable engine starts:

Requirements on engine start:

The first speed information must be available at the latest after 40 ms after the first crankshaft motion. This is necessary to be able to detect when the engine starts to rotate.

Requirements on engine stop:

- When switching off the engine,
 - "Powertrain: Engine Speed Including Start Stop" must deliver physically cor rect values down to 0 rpm.
 - there is a permissible max. delay time "c" between engine standstill and when the signal "Powertrain: Engine Speed Including Start Stop" is equal to 0 rpm.

- The engine is not permitted to switch off bus communication before "Powetrain: Engine Speed Including Start Stop" = 0 rpm.

5.4 Vehicle Motion relevant for Powertrain (VehMtnForPt)

This composition includes Powertrain functions related to vehicle motion. The following sections, 5.4.1 to 5.4.3 inclusive, define the components that have so far been agreed as part of this composition.

5.4.1 Driver Request (DrvReq)

Driver-specific conversion of accelerator pedal position to requested torque: determines the driver request related to the motion of the vehicle. For longitudinal motion, this functionality interprets the driver request as a torque request.



5.4.2 Accelerator Pedal Position (AccrPedIPosn)

The component calculates a percentage from the acquired position of the sensor and contains plausibility checks to ensure the information. Kick-down detection is included in this component.

5.4.3 Safety Vehicle Speed Limitation (VehSpdLimnForSfty)

Hard limitation of vehicle speed by engine torque reduction, without any comfort functionality.

5.4.4 Vehicle Motion (Powertrain): Miscellaneous (VehMtnForPtMisc)

VehMtnForPtMisc gathers together miscellaneous interfaces in the context of vehicle motion powertrain. The way in which these interfaces are used is not standardised. In future AUTOSAR releases, it is likely that these interfaces may be moved to different (more appropriate) provider or receiver components / compositions. It is even not excluded that they are moved to components that already exist.

VehMtnForPtMisc e.g. is used to close open interfaces in the case that it is committed that some component within vehicle motion powertrain will request or provide it but it is not yet decided which component or the component is missing.

5.5 Powertrain: Miscellaneous (PtMisc)

PtMisc gathers together miscellaneous powertrain interfaces. The way in which these interfaces are used is not standardised. In future AUTOSAR releases, it is likely that these interfaces may be moved to different (more appropriate) provider or receiver components / compositions. It is even not excluded that they are moved to components that already exist.

PtMisc e.g. is used to close open interfaces in the case that it is committed that some component within powertrain will request or provide it but it is not yet decided which component or the component is missing.



6 Additional Information

6.1 Differences between SW-Cs and ECUs

The SW components defined in chapter 4 are not to be confused with an ECU's functionalities.

For example, a combustion engine control ECU may contain the Combustion Engine SW-C plus other SW-Cs.

6.2 Functional safety

Many Powertrain signals are safety-relevant, therefore

- The AUTOSAR RTE will provide reliable communication for these signals at the low level, and
- Diagnostics and safety concepts for these signals must be applied at the higher, functional level.

AUTOSAR does not provide a Safety Concept for Powertrain systems. This must be done at the project level. This means that the specified interfaces must be checked to fulfill the safety requirements on each specific project.

6.3 Powertrain Application Interfaces - Decisions / Assumptions

6.3.1 Scope

In this document only passenger cars are considered.

6.3.2 PTC Composition (PtCoorr)

The PTC is not an atomic AUTOSAR SW-Component. In fact, its functionalities should be separated, into several sub-components. These sub-components will communicate with each other and with AUTOSAR SW-Components outside the PTC. The interfaces between the sub-components are not in the current scope, which is restricted to the definition of main interfaces between the non-PTC components and the PTC sub-components.

6.3.3 Definition of overboost

Overboost is a state in which the maximum torque which the combustion engine can deliver is increased for a limited period of time. Depending on the engine type, this could be realised, for example, as an increase in boost pressure on a turbocharged engine.



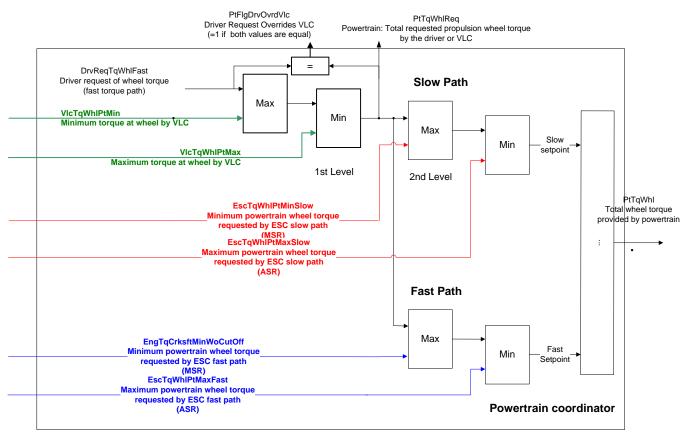
6.3.4 Coordination at the vehicle level

Coordination of vehicle energy (mechanical / electrical / thermal), vehicle operation modes, vehicle personalisation, etc., should be done at the vehicle level. This is not in the scope of the Powertrain Application Interfaces.

The composition VehMtnForPtMisc was added as an interim solution for some vehicle level issues relevant to the powertrain domain.

6.3.5 PTC Arbitration between Driver and Chassis torque requests

The two figures below show how the VLC and Stability Control torques requests could be arbitrated with the Driver Request. This is just an example to illustrate the concept behind the powertrain torque request interfaces, it is not intended to standardise the arbitration behavior in the PTC.

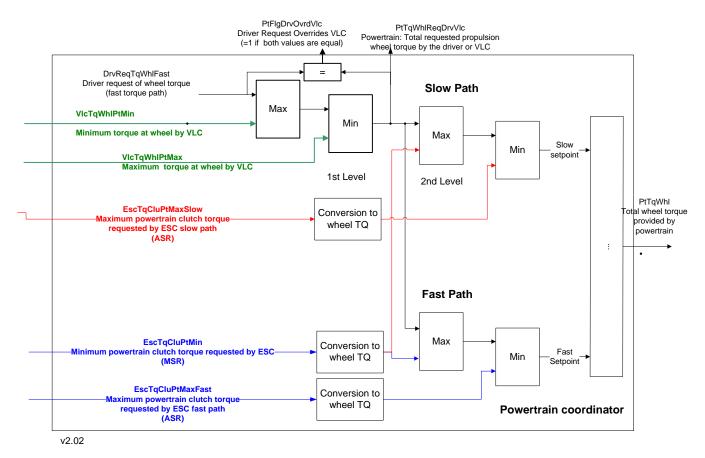


v2.02

VLC and ESP Arbitration in PTC



Explanation of Application Interfaces of the Powertrain Engine Domain AUTOSAR CP R20-11



VLC and ESP Arbitration in PTC based on TQ at clutch

6.3.6 Assumptions on modeling style and naming aspects specific for powertrain domain

AUTOSAR provides a guideline for modeling and naming of model elements ([1]).

There are architectural design patterns like the sensor actuator design pattern described in [11] that also include modeling and naming aspects.

In this section only additional patterns and modeling styles followed are explained to get an overall understanding of the signals standardized for the powertrain domain.

Please note: Here standardized ports or port interfaces mean standardized port prototype blueprints or port interface blueprints [7].

Kind of Modeling in general applied within powertrain domain, especially if the sys- tem modeling guideline [1] gives some freedom.		
Kind of modeling or as- Rationale		
sumptions		
All SenderReceiverInterfaces	In earlier versions the standard did not contain in-	
standardized are assumed to be measurable.	formation about calibration and measurement.	
	Since R4.0.3 all data types allow measurement by	
	default (see generated .arxml [3]). So our implicit	



Kind of Modeling in general applied within powertrain domain, especially if the system modeling guideline [1] gives some freedom.			
Kind of modeling or as- sumptions	Rationale		
	assumption that all signals are measurable is ful- filled.		
All ports are assumed to be optional.	Within our example components all ports are as- sumed to be optional. The ports are derived from the port prototype blueprints with the same name. It is optional per default that port prototype blueprints are allowed to be used but not necessarily used in every project.		
	In previous releases without blueprints this assump- tion was very important because there was no vari- ant handling done. So within the powertrain domain it was assumed that all ports are optional. Since only ports but no components are standard- ized this was even more important: it means that a supplier or OEM may create a single SW-C (Soft- ware Component) and use only the standardized ports that are relevant for this SW-C in his sw archi- tecture.		
Port interfaces are not de- signed to be reused: there is a 1:1 relationship between port and port interface. The port interface has the same name as the port + an addi- tional index as required by the System Modeling Guideline [1]. Exception: If powertrain is not the provid- er then the rules of other do- mains are respected.	Ports are attached to SW-C. Since SW-C are not standardized only port interfaces were really subject of usage in projects up to Release 3.1. With Release 4.0 the standardization of ports is sup- ported by using so-called <i>PortPrototypeBlueprints</i> in the meta model. However, in practice older versions of the AUTOSAR meta model are still in use and the exist- ing tools do not yet fully support <i>PortPrototypeBlue- prints</i> . To be backward compatible and to enable the easy introduction of the standardized application interfac- es within the powertrain domain not all features of the meta model (like e.g. connectiong of compatible interfaces with different port interface short names) therefore were yet fully exploited. Connecting of compatible interfaces with different port interface short names is not supported.		
If a port interface contains exactly one data prototype its name is identical to the port interface excluding the trailing index.	There were only two alternatives: - using full name - using name "Val" for Value Disadvantage of solution 2) would have been that many ports would have been assumed to be com- patible by tools because identical data prototype names (with compatible interfaces) allow an auto- matic connection. Within specification tools like e.g. ASCET-SD or		



Kind of Modeling in general applied within powertrain domain, especially if the system modeling guideline [1] gives some freedom.			
Kind of modeling or as- sumptions	Rationale		
	MATLAB/Simulink it might be possible to only show the data prototype name and to not display the port interface or port name.		
The reuse of data types was explicit goal within the power- train domain.	Within the powertrain domain it was an important design goal to use as few data types as possible.		
Computation methods were not subject to reuse yet.	The reuse of computation methods is not supported, but only re-use of data types. But reuse of computa- tion methods should be considered for the next re- lease.		
In most cases only one data prototype was defined per port interface.	This kind of modeling allows the biggest flexibility in implementing the standard.		
	On assembly level e.g. it is not allowed that the r- port has more data prototypes than the p-ports ([6], figure 6.3). Therefore in such cases, when several data prototypes are part of one r-port the data proto- types themselves cannot be assumed to be optional, only the complete r-port.		
	In older versions of the AUTOSAR standard is was not allowed that a sub-component provides only part of the port, i.e. if several data prototypes were part of a port interface it was implicitly standardized that there is one SW-C providing it. When splitting the information on several port interfaces each data pro- totype might be provided by a different SW-C.		
In most cases no records were used to define port inter- faces.	The rationale is similar to the one stated for the as- sumption to only define one <i>DataPrototype</i> per <i>Port-Interface</i> . Since within the powertrain domain <i>PortInterfaces</i> with multiple <i>DataPrototypes</i> are seldomly used, there is no necessity to use records and it is as- sumed that timing related aspects of the data proto- types are to be handled separately, that there are other more flexible possibilities to do so.		

Specific Naming Assumptions for Powertrain Domain	
Assumption	Rationale
The names were chosen such that automatic generation of display names is possible.	In Powertrain ECUs there are thousands of calibra- tion relevant data. So it is important to apply the System Modeling Guideline [1] in a way that auto- matic generation of display names is possible. See [5] for details.
In general the keyword abbre- viation "Consold" for "Consoli-	Reason for doing so is to have short names when- ever possible (see [TR_MCM_70020] in [5])



Specific Naming Assumptions for Powertrain Domain		
Assumption	Rationale	
dated" as well as the abbrevia- tion "Act" for "Actual" was sup- pressed.	Example: <i>PtTqClu</i> for "Torque at Clutch "	
In case a port is only for Fast Path or Slow Path "Fast" and "Slow" is added. In the other	Reason for doing so is to have short names when- ever possible (see [TR_MCM_70020] in [5]) Example:	
cases "SlowFast" is suppres- sed.	<i>PtTqWhlMinWoCutOff</i> affects Slow and Fast path <i>EscTqWhlPtMaxSlow</i> or <i>EscTqWhlPtMaxFast</i> is for a specific path	
	For more details on fast and slow paths see chapter 3.3	
"State" is used as abbreviation "St" for "State" and "Status".	In most cases it is very difficult to explain the differ- ence between State and Status. So for sake of simplicity and consistency within the powertrain domain only "St" is used.	
TqWhI means sum of Torque Wheels whereas TqWhIInd means the torque of an indi- vidual wheel.	Within Powertrain the sum of torque wheels is more often used. So information ("Ind") is added in the case that individual wheel torques are meant. Example:	
	<i>PtTqWhl</i> for "Total Wheel Torque Provided by Powertrain" <i>PtTqWhlInd</i> for "Torque Delivered by the Power-	
	train to the Individual Wheels"	
Prepositions are only used in the short name if it really helps understanding.	In most cases the preposition does not really add information and makes short names unnecessarily long (see [TR_MCM_70020] in [5]).	
	Examples: <i>PtTqWhIMinWoCutOff</i> for Minimum Possible Total Powertrain Torque at Wheels Without Complete Fuel Cut Off using the preposition 'Wo' (without)	
	<i>PtTqWhIReqDrvVlc</i> for Powertrain: Total Requested Propulsion Wheel Torque by the Driver or Vehicle Longitudinal Control (VLC) instead of	
	<i>PtTqAtWhIReqByDrvForVlc</i> as recommended by the system modeling guideline [1].	
Special rules for engine- transmission interfaces	 Each Long Name starts with the provider and co- lon ":". Only if the provider cannot be defined, the provider and the colon are omitted. 	
	 If the reference object and the reference level for the speed are not explicitly stated, then the refer- ence object and the reference level for the speed 	



Specific Naming Assumptions for Powertrain Domain	
Assumption	Rationale
	are the object itself.
	Example:
	a) "Engine: Torque at Crankshaft" is the torque at
	the current crankshaft speed, not at idle speed,
	max. speed,
	b) "Engine: Torque at Crankshaft" is the torque at
	the crankshaft level, not at wheel level.